



Latest Advancements in DrillString Mechanics

SPE Gulf Coast Section – 03/09/2016

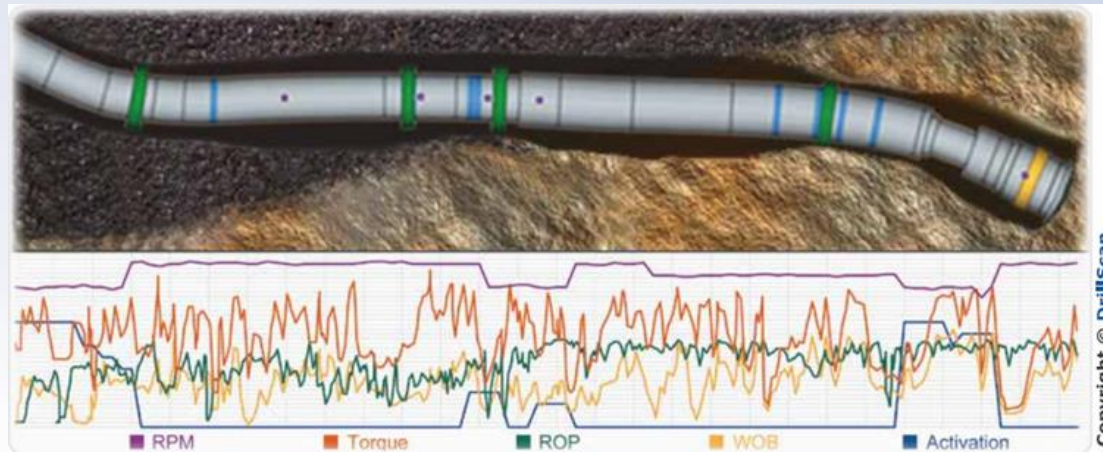
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- **DrillScan Intro**
- **Directional Drilling**
 - Problem Statement
 - Bit Steerability
 - Walking Tendency
 - Global vs Local Curvature
 - Unconventional Well Example
- **Torque & Drag & Buckling**
 - Soft versus Stiff String
 - Buckling Theory
 - Lab and Field Validation
 - Unconventional Well Example
- **Casing Wear**
 - Problem Statement
 - New Casing Wear model
 - Field Validation
- **Conclusion**

- **Expert Services, Innovative Software Solutions, Trainings for the drilling industry**
 - Directional Drilling, Torque & Drag & Buckling, Survey, Casing Wear, Fatigue, Drilling Bit Performance, Drilling Dynamics
- **Advanced Modeling Solutions**
- **Strong collaboration with Research**
 - Laboratory Validation & Permanent improvement
- **Strong collaboration with Operators**
 - Field Validation



The directional behaviour of any drilling system depends mainly on:

- **The Directional System:**

- Rotary Steerable System (RSS)
- BHA rotary
- Steerable Mud Motor
- With/without Reamer Capability



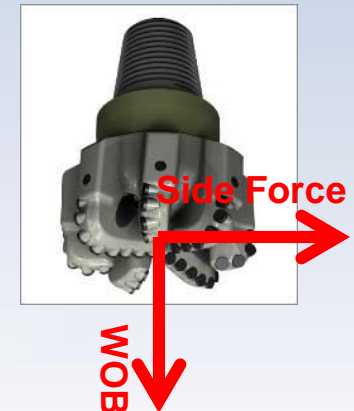
- **The Rock Formation:**

- Hardness (UCS)
- Anisotropy (dip angle)



- **The Drilling Bit Characteristics**

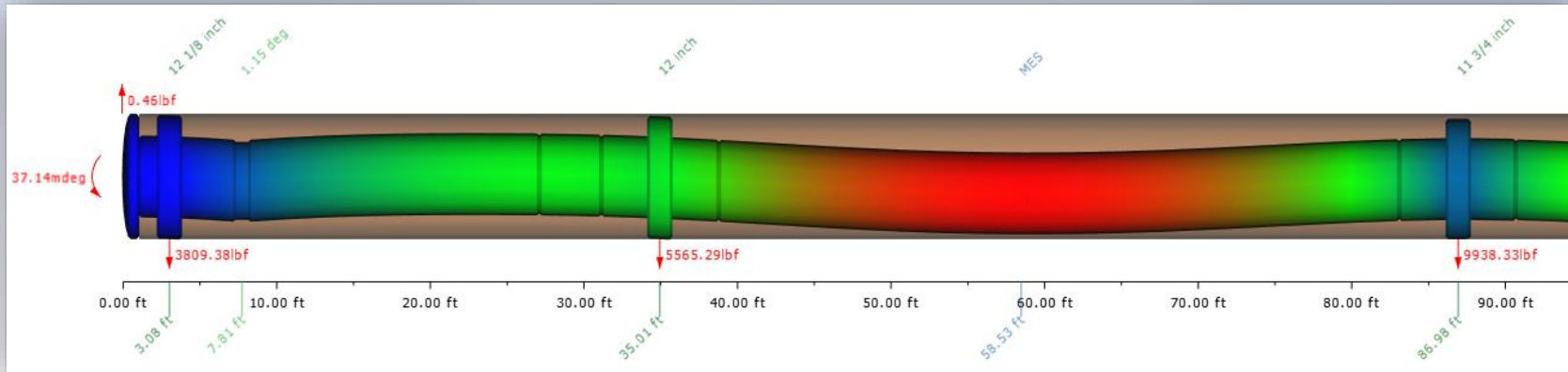
- Walking tendency (Turn rate)
- Steerability = Side-cutting ability (Build/Drop Rate)



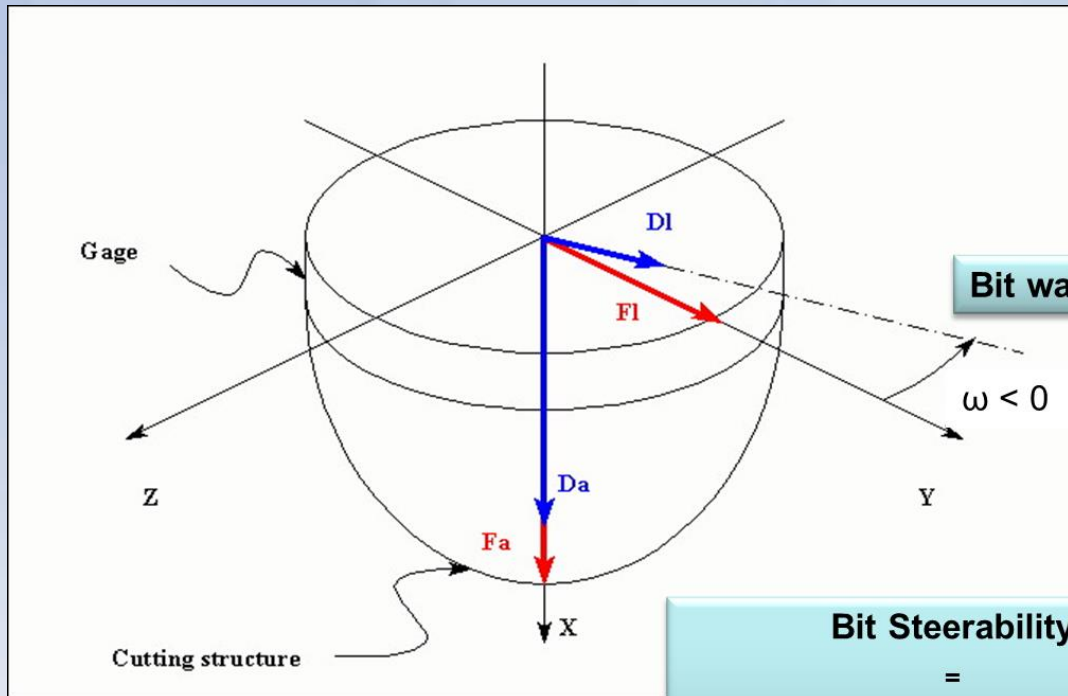
Rock-Bit model



BHA model



SPE 74459, PA-82412, 79795, PA-87837, 110432



Bit walk angle

Turn
Rate

$$\begin{aligned} \text{Bit Steerability} &= \\ &= \text{Lateral Drillability} / \text{Axial Drillability} \\ &= \\ &= (DI/FI) / (Da/Fa) \end{aligned}$$

Build/Drop
Rate

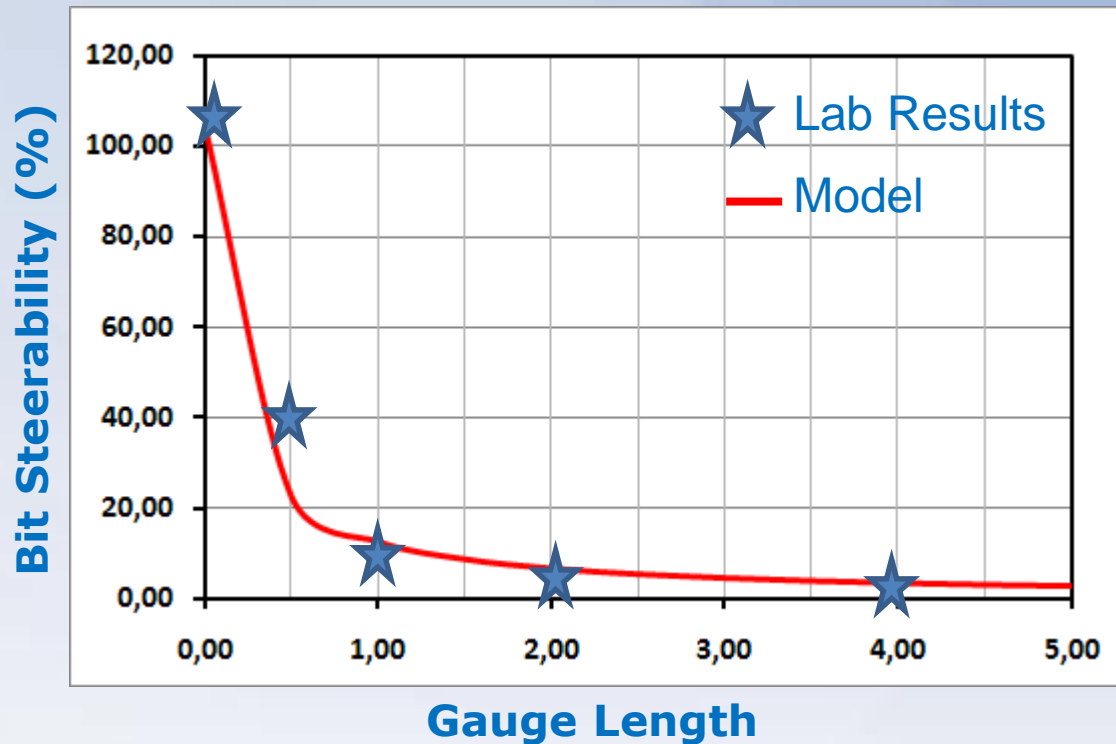
High Bit Steerability = High Side-Cutting ability of the bit

Bit Steerability = 5 - 50% for most PDC Bits

Effect of Gauge Length on Bit Steerability

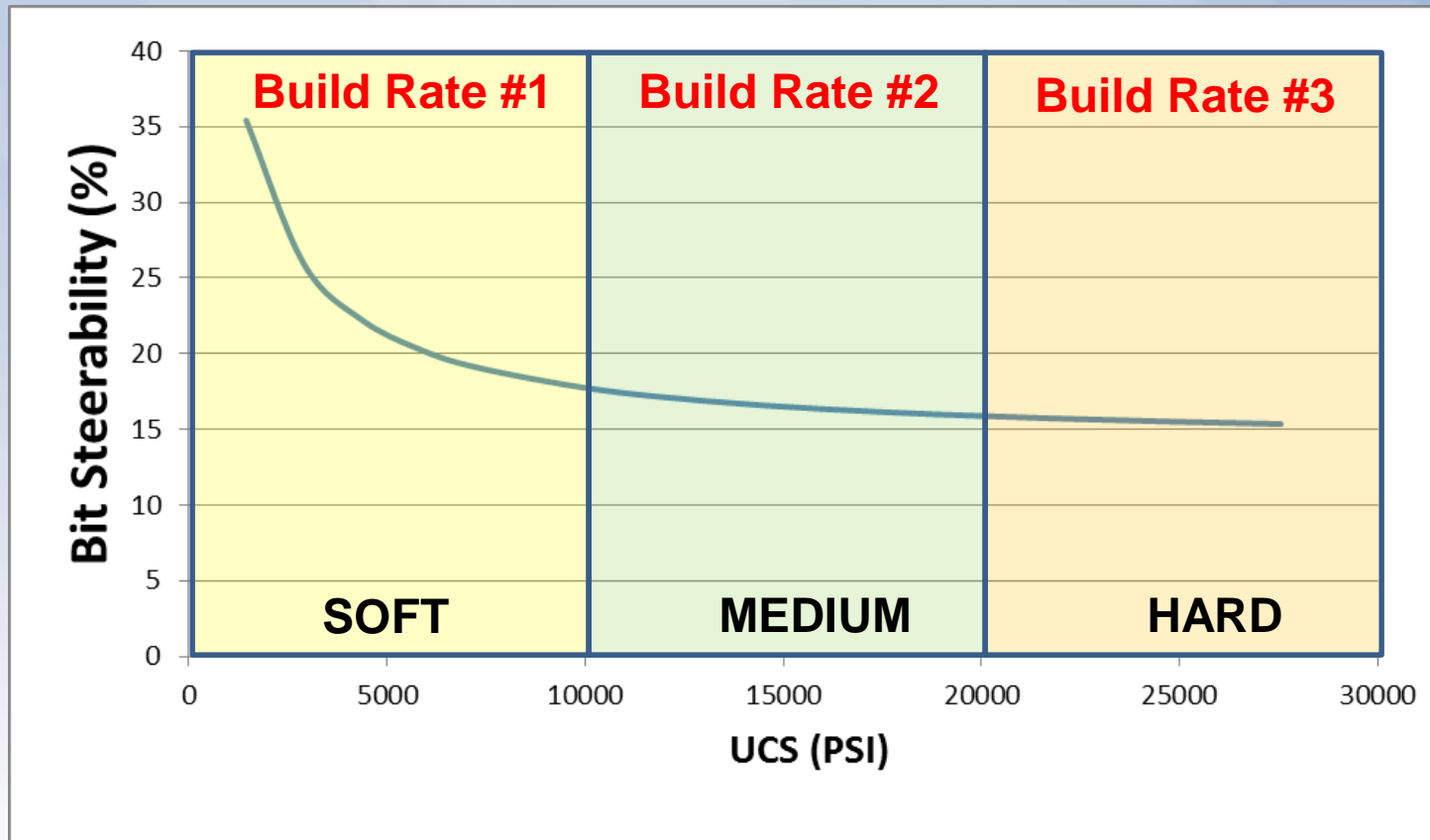


Gauge Length



SPE 74459, PA-82412, 79795, PA-87837, 110432, 151283

Effect of Rock Hardness



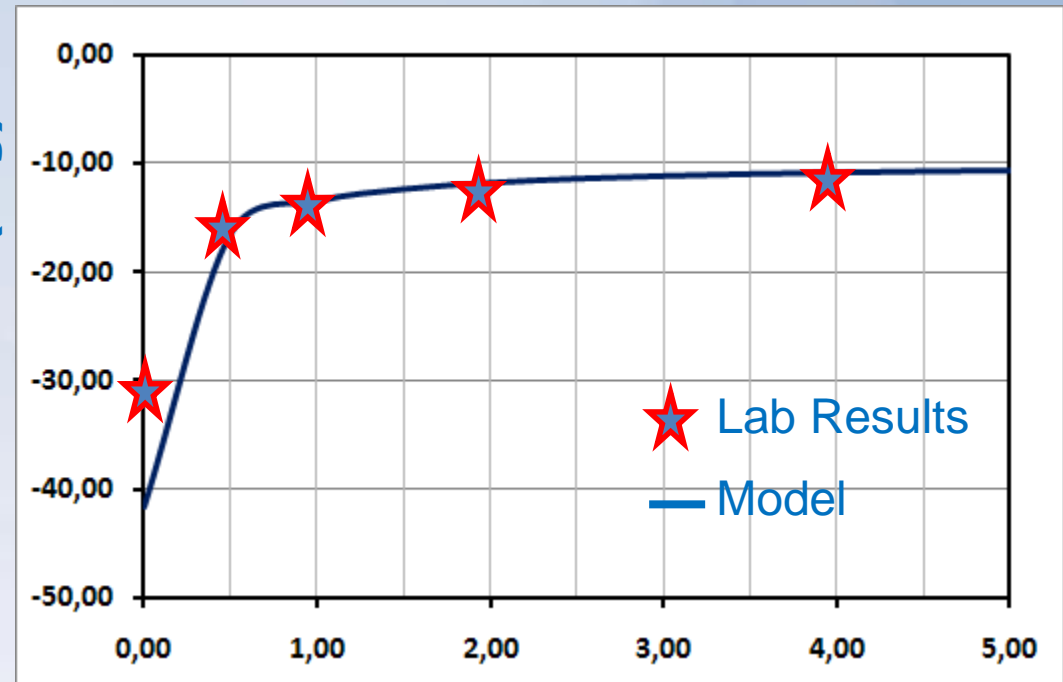
Higher Side-Cutting in a Soft Formation

Effect of Gauge Length on Bit Walk Angle



Gauge Length

Bit Walk (deg)



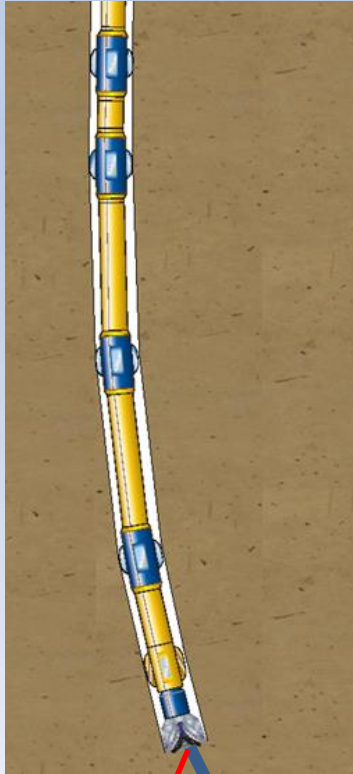
Gauge Length (inch)

$\tan (12 \text{ deg.}) = 0.21 \gg$ Coefficient of friction steel-rock

Generally speaking: if the coef. Of friction \nearrow Bit Walk \nearrow Turn Rate \nearrow

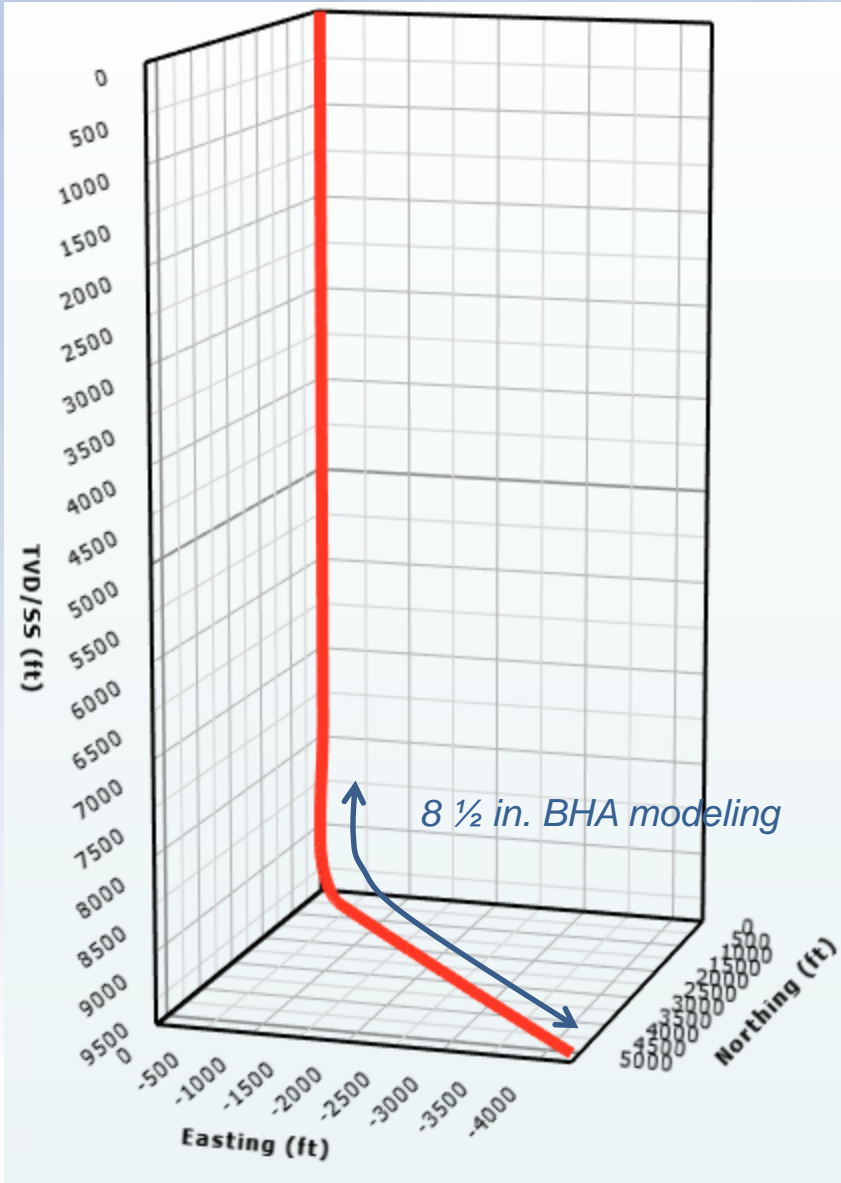
2 Methods:

- **Equilibrium curvature**
 - Global response over 100 ft or so
 - Global Directional Objective
- **Step by step**
 - Local response over 5 ft or so
 - Tortuosity
 - Hole Quality



Required Data:

- **Well Trajectory**
- **BHA details:** ID, OD, Bend angle & position, Stabilizers, etc...
- **PDC bit specs:** Gage length, Bit Profile
- **Sliding/Steering sheet:** TFO, slide/rotate, activation level (RSS)
- **Mud weight**
- **Operating Parameters:** WOB, RPM
- **Rock:** Unconfined compressive strength (UCS)



- **Rock**

- UCS = 7000 psi

- **8 1/2 in. PDC Bit**

- 2 inch Gauge Pad
- Bit Steerability = 6%
- Walk angle = -12 deg.



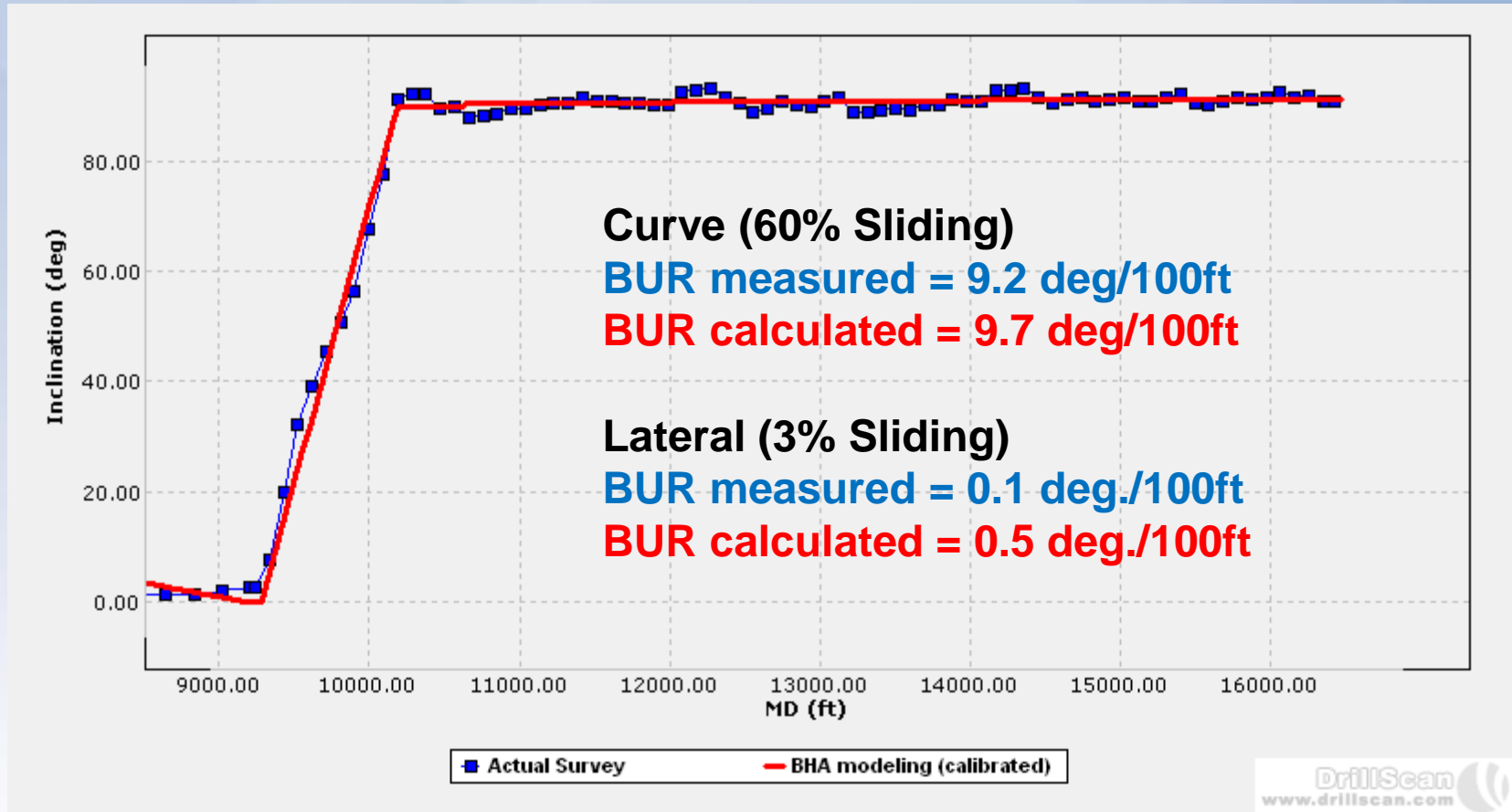
- **BHA**

- Slick Assembly. 2 deg. bend
- 7 in. 5/6 lobes Mud Motor

- **BHA modeling**

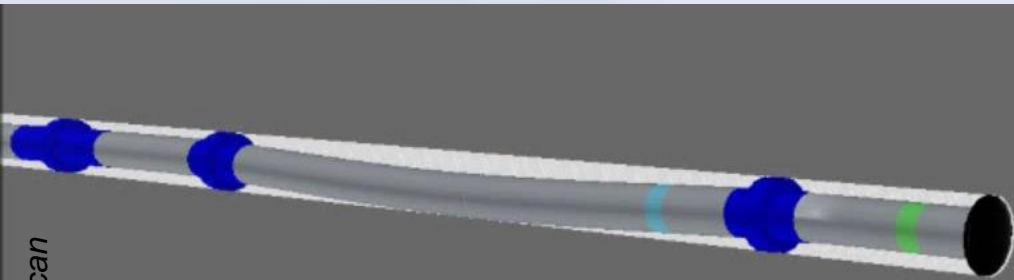
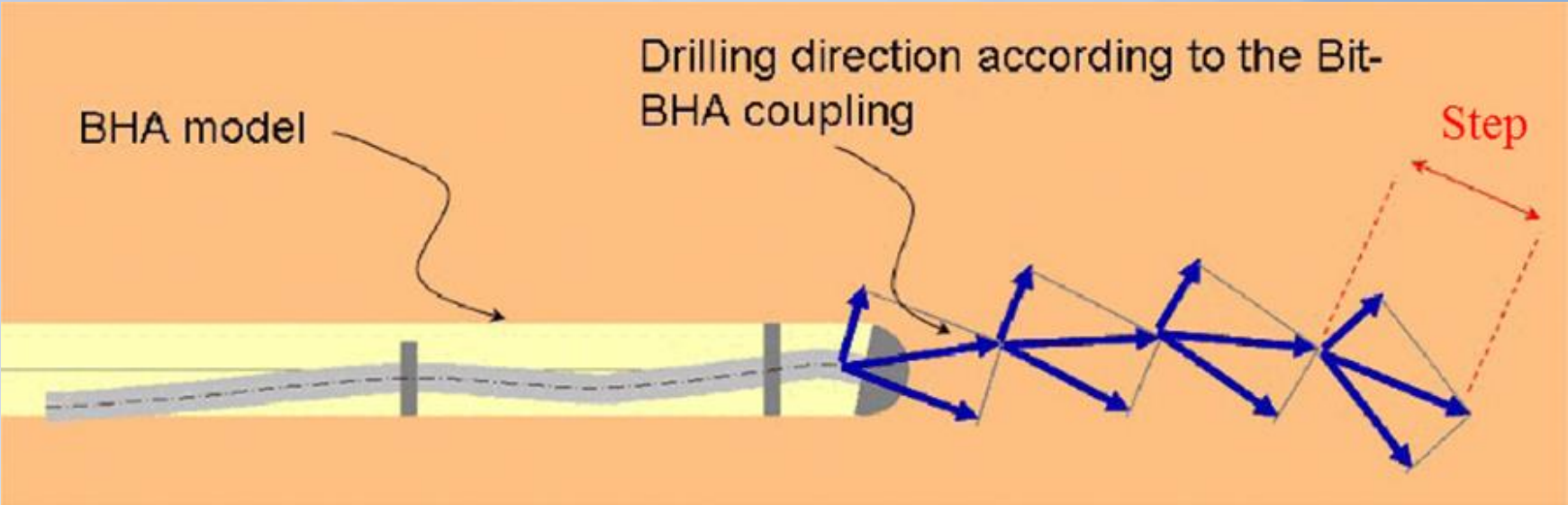
- Curve + Lateral

Equilibrium approach = Global Curvature

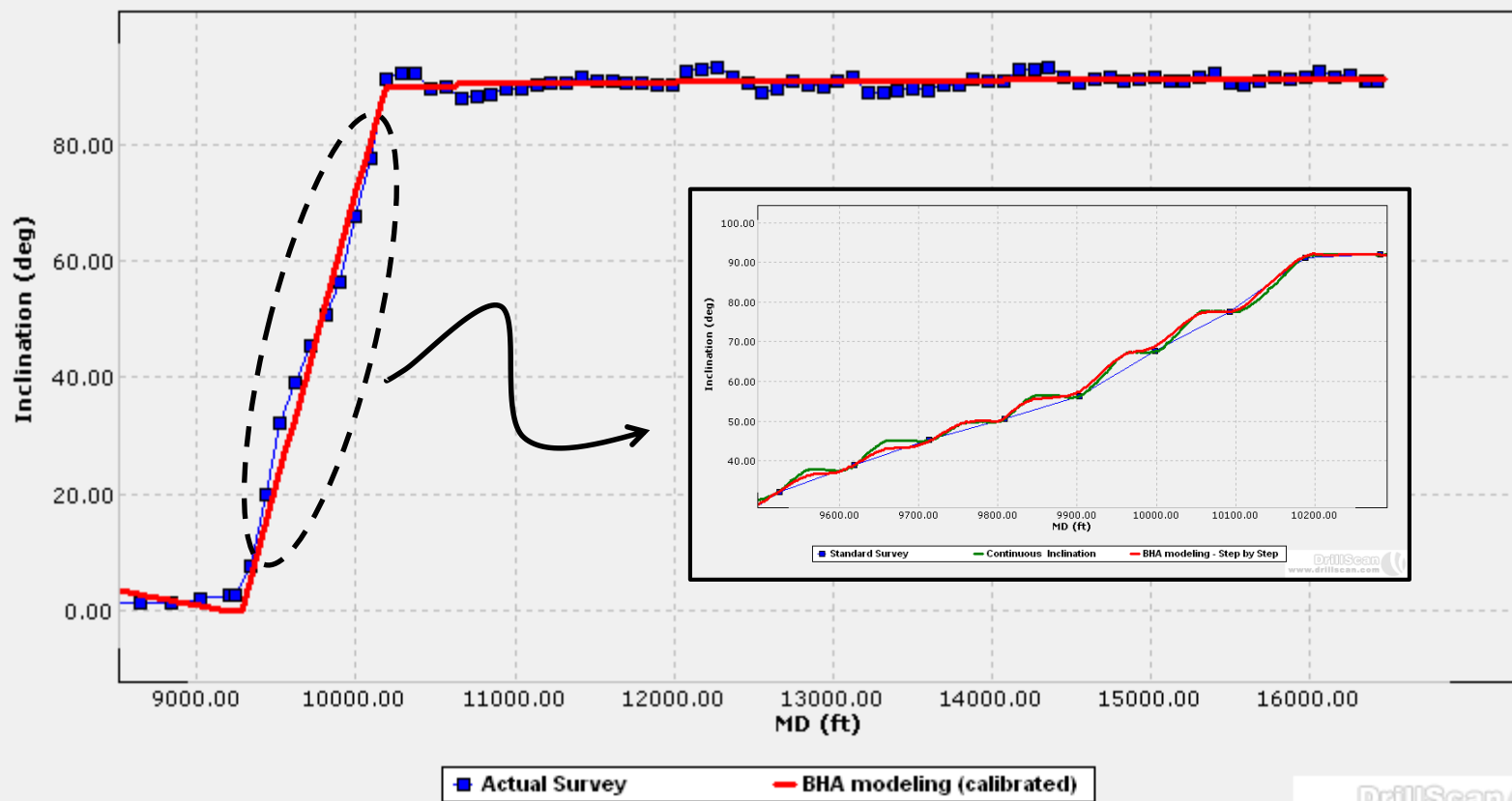


Reduction of Sliding in the Lateral Section >> Neutral BHA

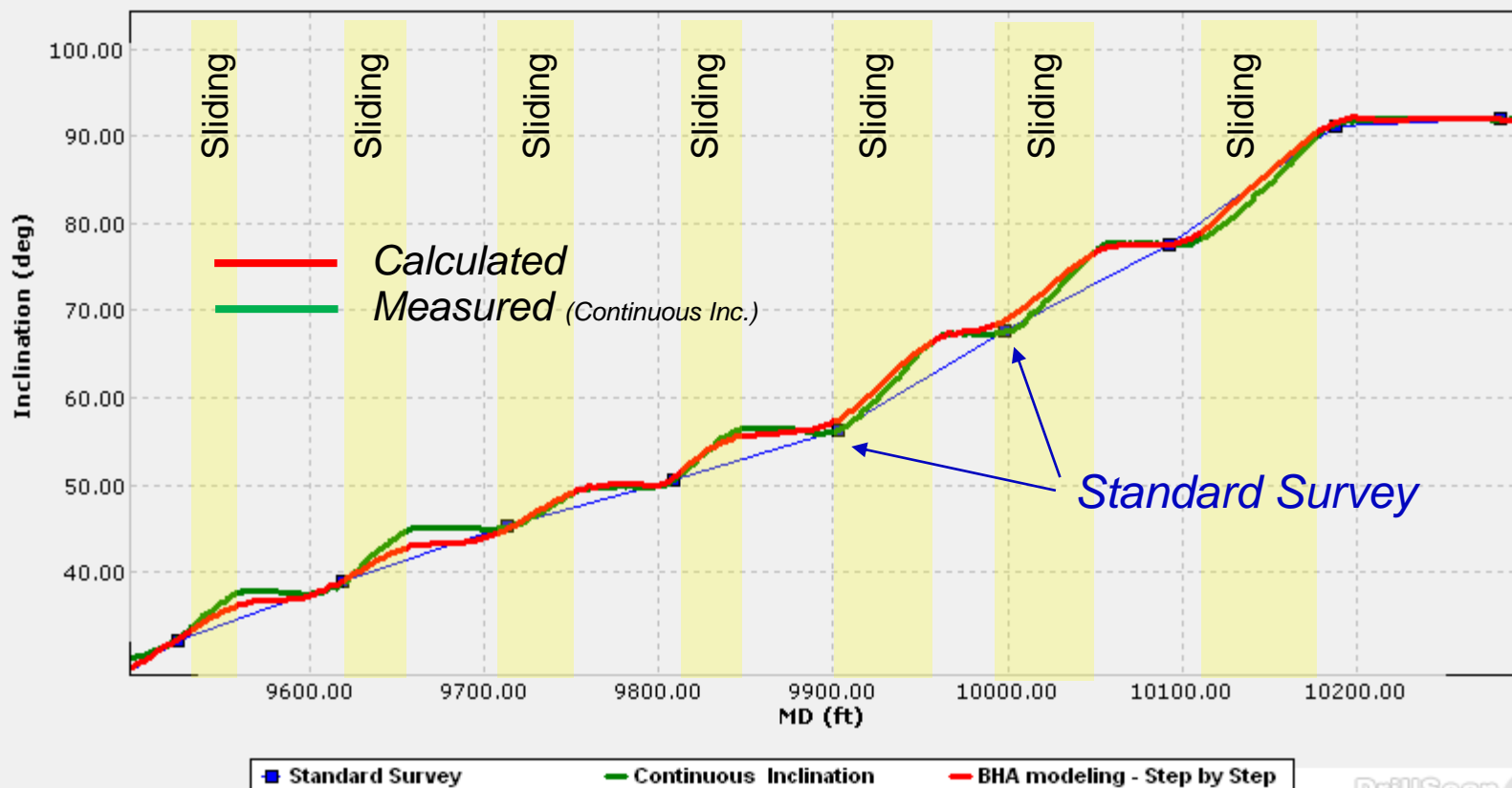
If Slick Assembly = Gauge Length & WOB play a great role to make the BHA neutral



From Global Curvature to Local Dog Legs



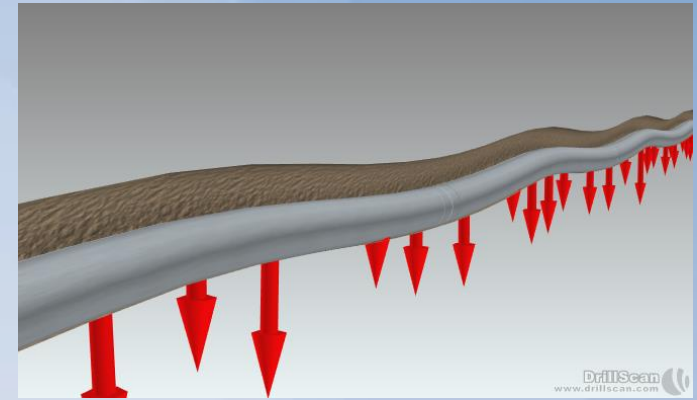
Step by Step Calculation vs Continuous Inclination Measurements



- Step by Step calculation
 - Borehole Tortuosity Evaluation
 - RSS / Steerable Mud Motor / BHA rotary
 - Fine tuning of the BHA to reduce tortuosity
 - Better Torque & Drag Prediction
 - More realistic tortuosity
 - Better Wellbore Placement
 - About 20 ft difference in TVD between Standard Survey vs Continuous Survey

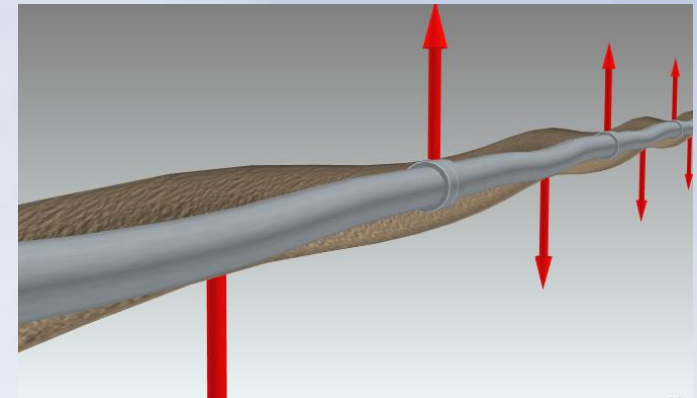
- Soft-string model

- Johancsick et al. (1983)
- No Stiffness (it's a cable)
- Continuous contact on the low side of the borehole



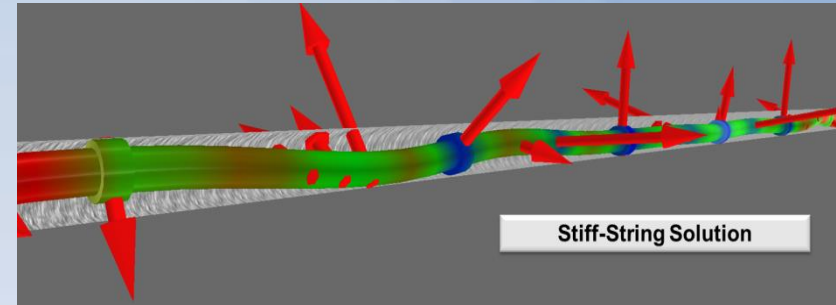
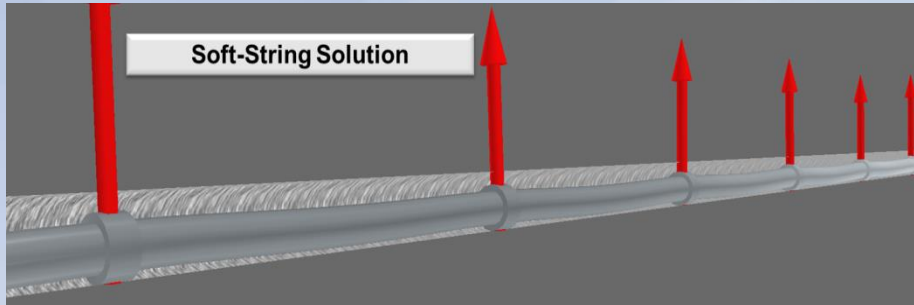
- Stiff-string model

- In collaboration with
- Stiffness
- Unknown contact points computation

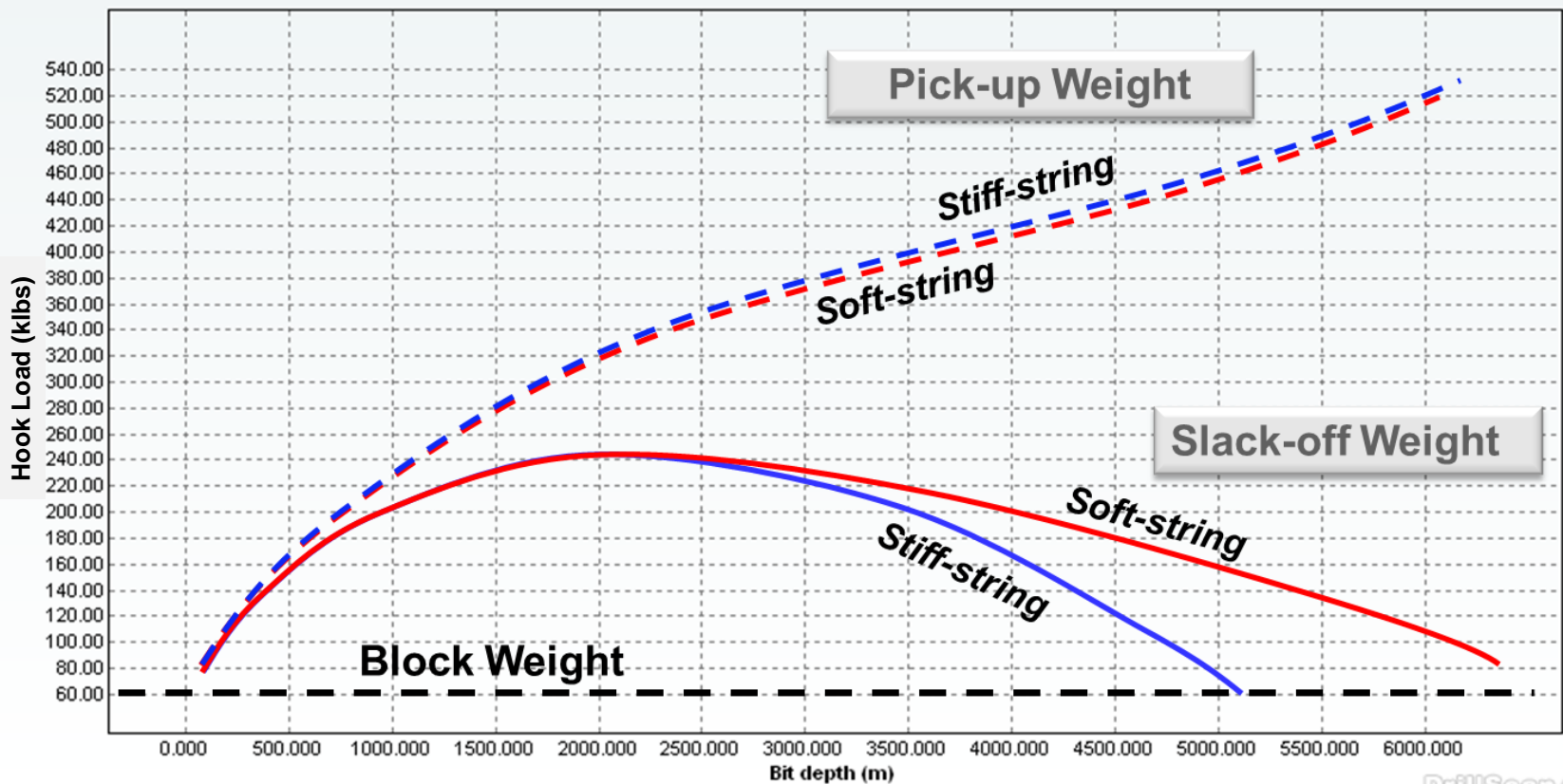


- **3D Stiff-string**
- **Fundamentals : Mines ParisTech**
 - **SPE 98965, SPE 102850-PA (modeling details), SPE 112571**
 - **SPE 119861, SPE 140211, SPE 151279**
- **Without FEA (Computation Time Reduced)**
- **Powerful Drillstring-Hole Interaction Contact Calculation**
- **Only provider of Simultaneous Torque-Drag-Buckling Calculation**
- **Any Type of Tubular Handled (beam element in 3D space)**
- **Hole Size and Clearance Effects**
- **Micro and Macro-Tortuosity Effects**





Engineering Features	Soft-string	Stiff-string
Clearance / Hole Size	✗	✓
Stiffness / Bending	✗	✓
Contact Calculation	✓ ✗	✓
Post-Buckling Calculation	✗	✓
Mechanical Integrity	✗	✓



Up to 5% difference in PUW

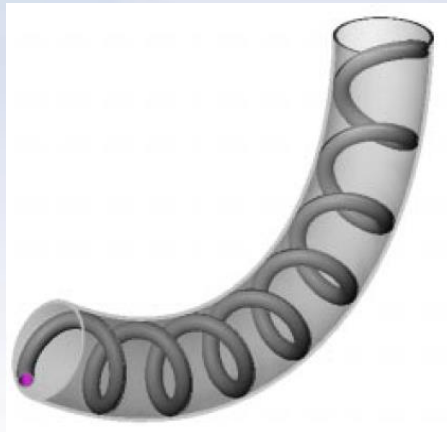
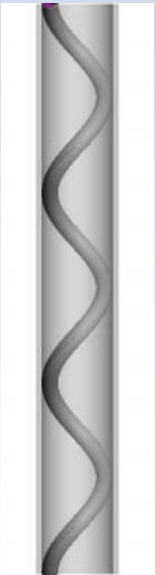
Up to 20% difference in SOW

Up to 30% difference in Torque

Up to 50% difference for Post-Buckling Calculation

Sinusoidal

$$F_c = 2 \sqrt{\frac{EI \omega \sin(Inc)}{r}}$$



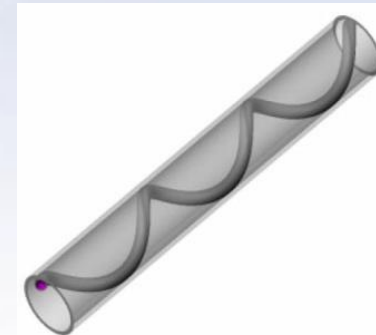
Helical

$$F_c = \lambda \sqrt{\frac{EI \omega \sin(Inc)}{r}}$$

$$\lambda = 2\sqrt{2} = 2.83 \dots \dots \dots \text{Chen \& Cheatham}$$

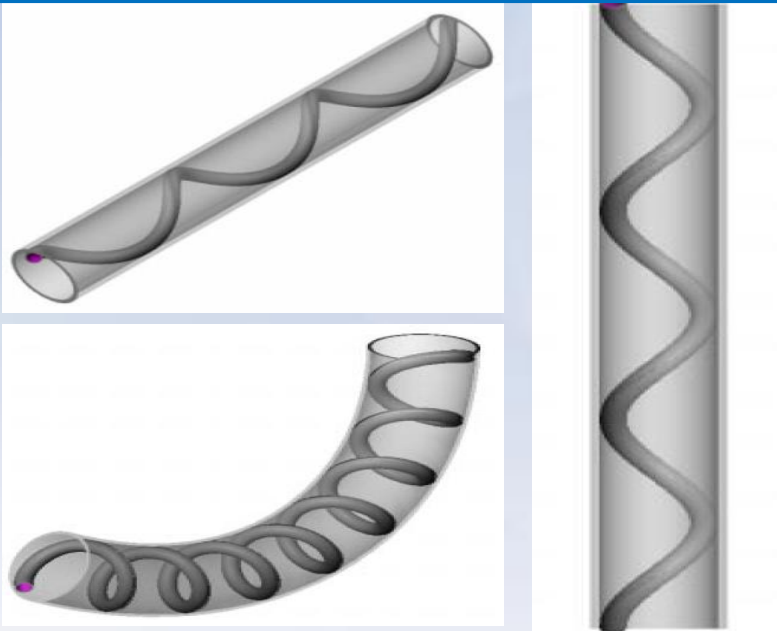
$$\lambda = 2(2\sqrt{2} - 1) = 3.65 \dots \dots \dots \text{Dawson \& Paslay}$$

$$\lambda = 4\sqrt{2} = 5.65 \dots \dots \dots \text{Mitchell}$$



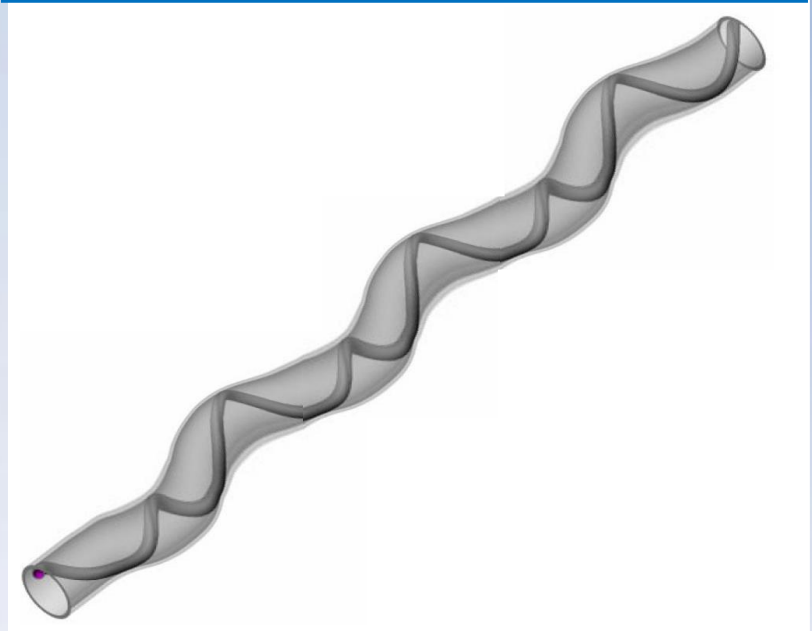
Standard Buckling Criteria

Idealized Case



Advanced Buckling Modeling

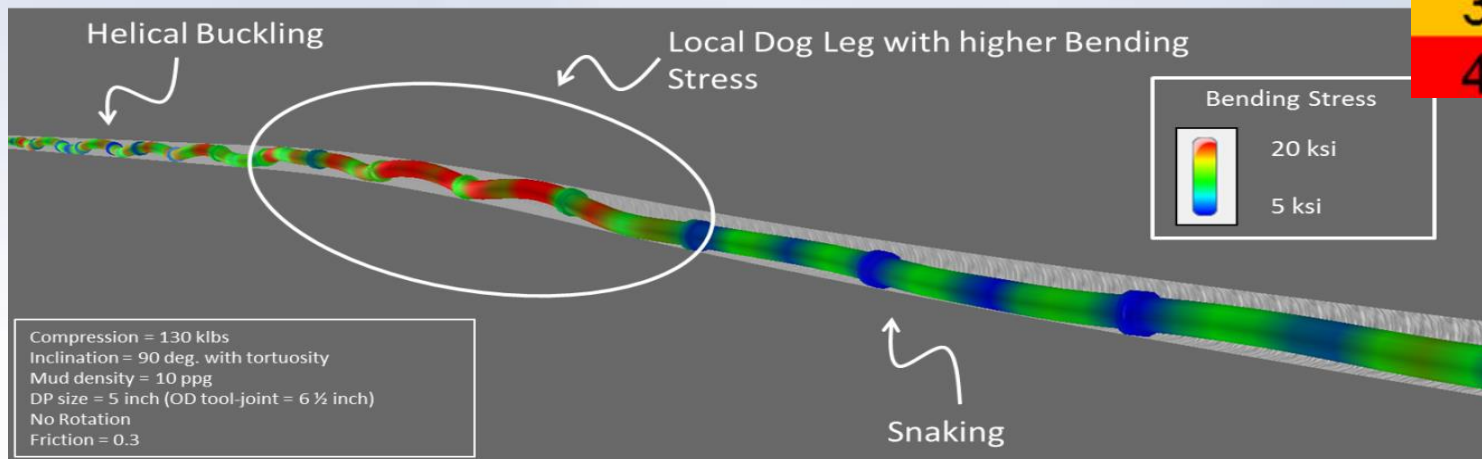
Field Conditions



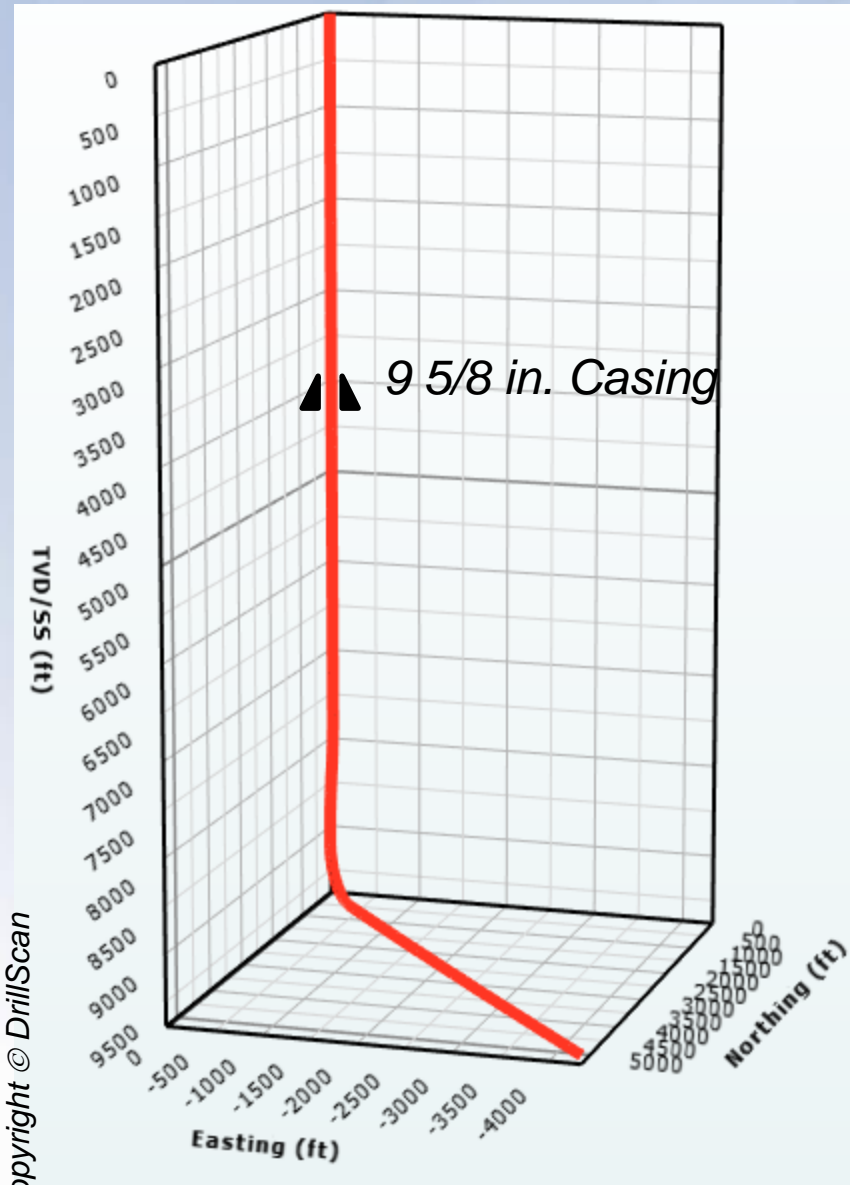
Rotation, Friction and Dog Legs have a great effect on Buckling

New Buckling Criteria: Buckling Severity Index

- Laboratory and Field evidences have shown that standard Buckling Theories fail sometimes to predict Buckling
 - Ref: SPE 102850, SPE 112571, SPE 119861
- Drilling or tripping in the hole in exceeding standard buckling loads is still possible (reasonable bending stress level): Shale Gas Wells
- New criterion based on the pipe stress rather than the pipe shape
- **Buckling Severity Index (BSI)**
 - Ref: SPE 151279, SPE 151283



- | | |
|---|-------------|
| 1 | Low Risk |
| 2 | Medium Risk |
| 3 | High Risk |
| 4 | Severe Risk |



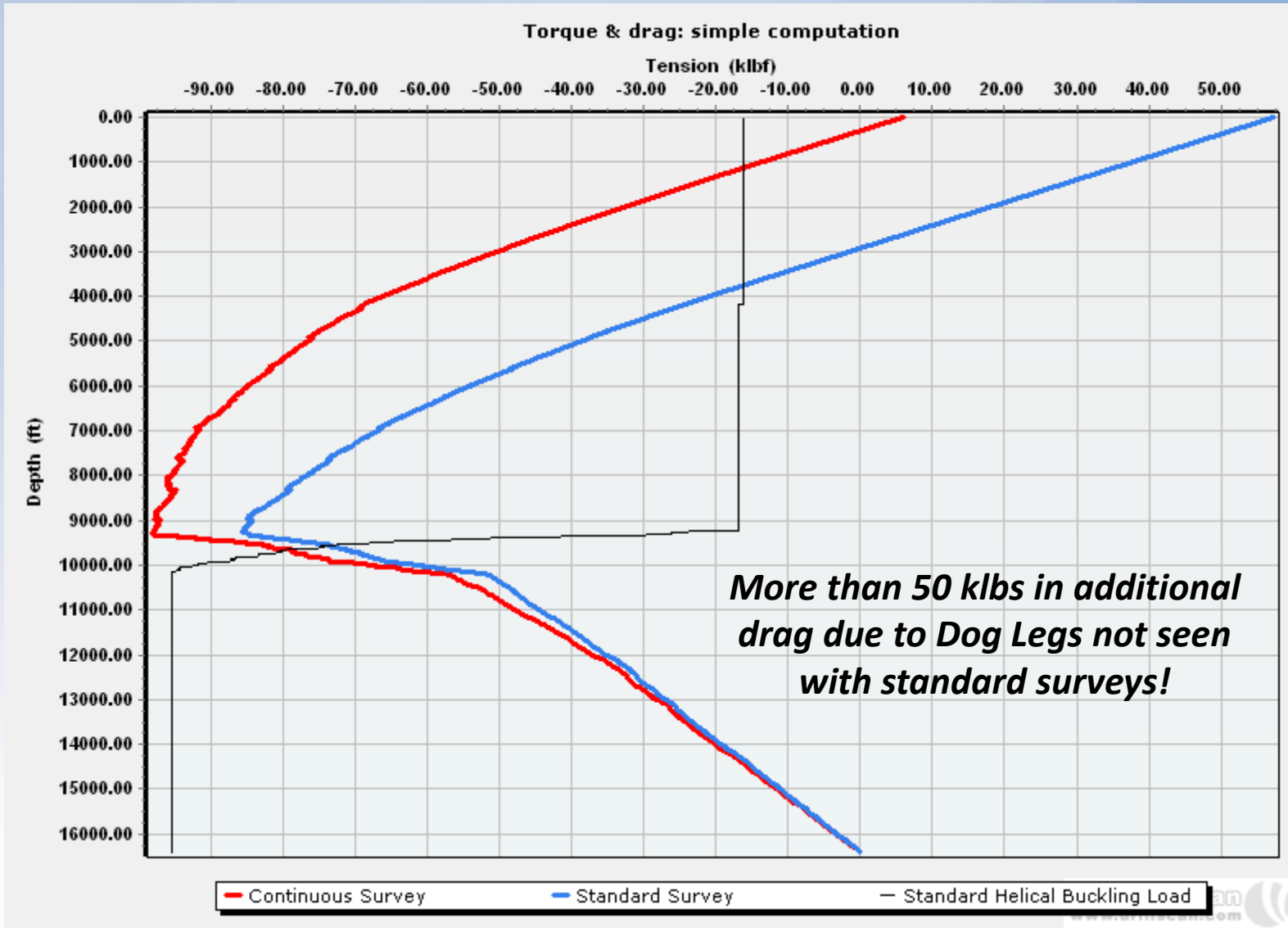
• Run In Hole Simulation

- 5 ½ Casing String
- Linear Weight = 23 ppf
- Mud weight = 11 ppg
- Coefficient of Friction
 - 0.20 in Cased Hole
 - 0.38 in Open Hole

• Comparison

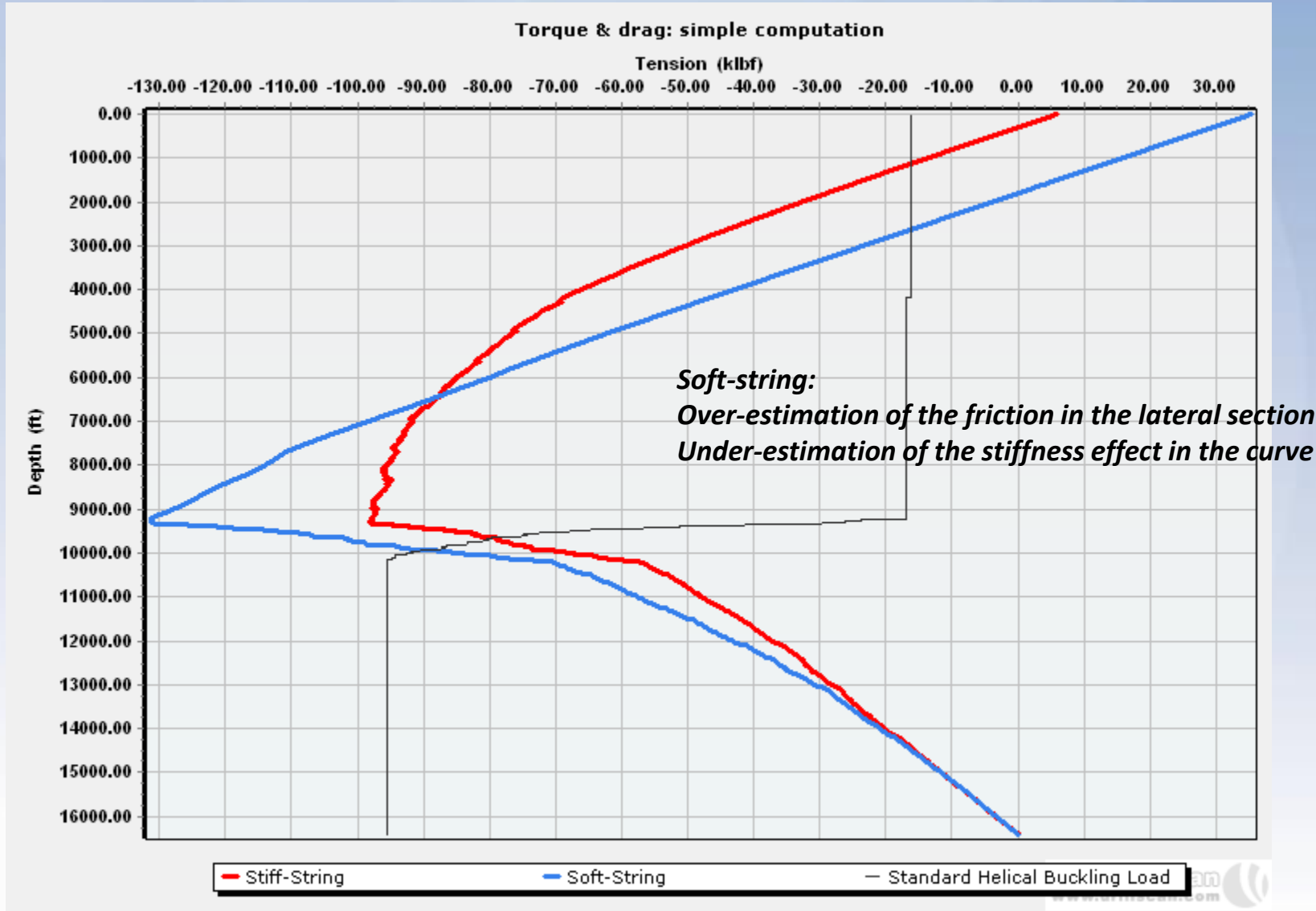
- Standard vs Continuous Survey
- Soft-string vs Stiff-string
- New Buckling Severity Index

Stiff-String Model



Stiff-String vs Soft-String Model Continuous Surveys

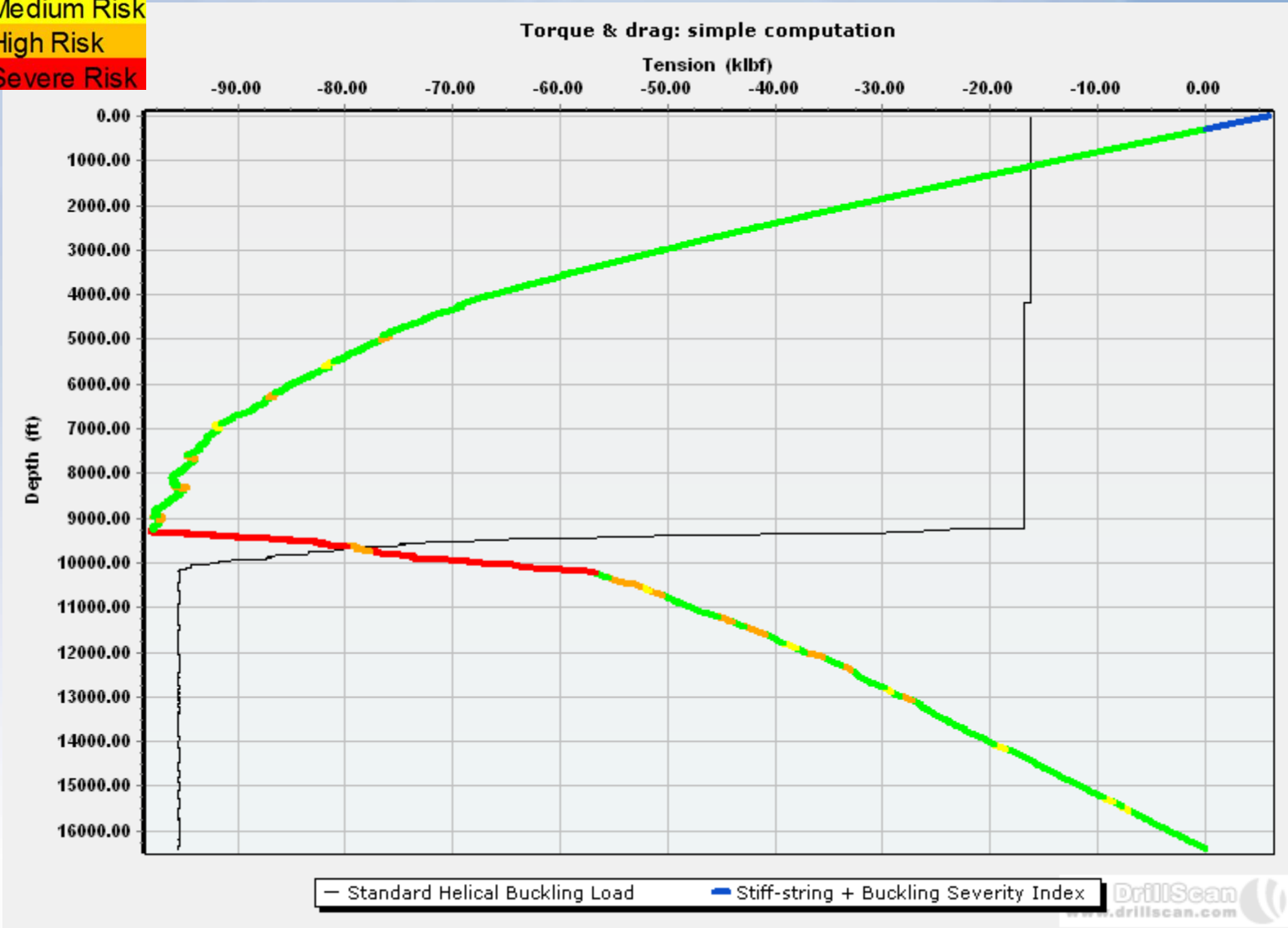
Case Study: Unconventional Well



Buckling Severity Index

- 1 Low Risk
- 2 Medium Risk
- 3 High Risk
- 4 Severe Risk

Case Study: Unconventional Well



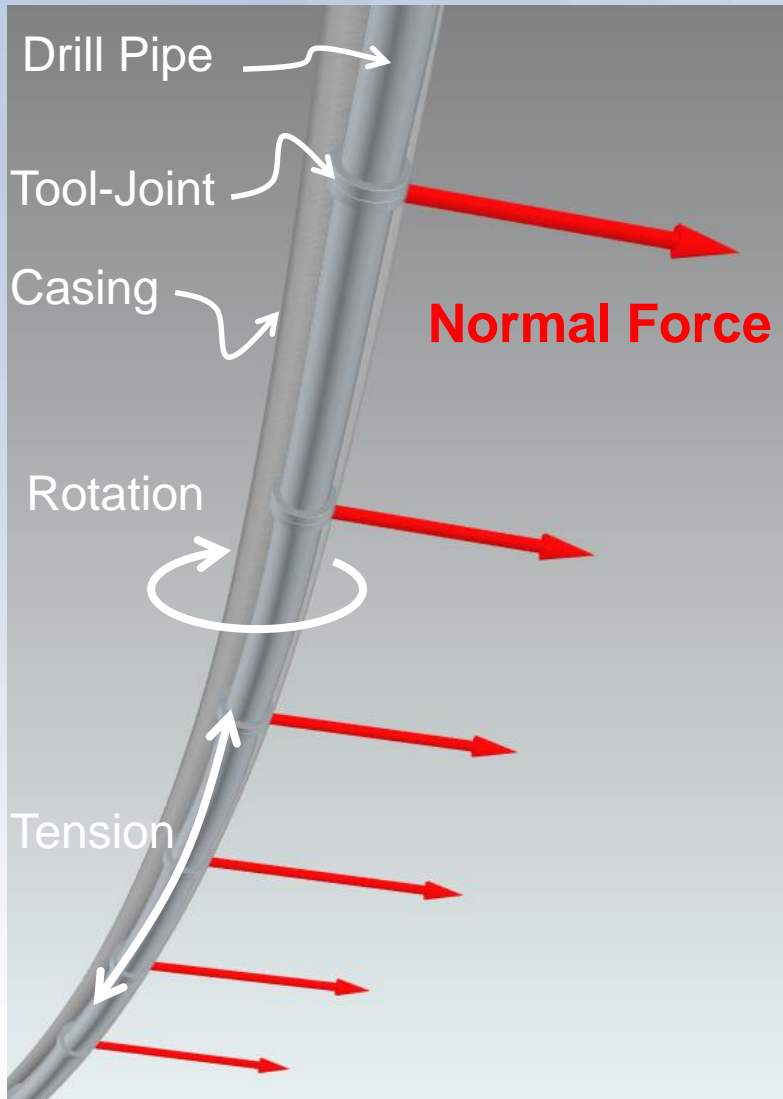
Result: RIH@TD Cont. Inc. 0.38
Well: Well 03
Trajectory: Cont. Inc
Casing program: CSG
String: 5.5in CSG No cent.
MD: 8879.23 ft
Inc: 1.40 deg
Azi: 171.02 deg
DLS: 0.52 deg/100ft
Contact force: - lbf
Contact force
4824.00 lbf
0.00 lbf
Bending stress: 9081.00 psi
Bending stress
18000.00 psi
0.00 psi
Tension: -97.41 klbf
Torque: -60.98 lbf.ft
Displacement: 1.34 inch

Zone of helical buckling but with Low Bending Stress

Zone with higher bending stress (due to high dog legs)

9000

Problem Statement



Casing Wear

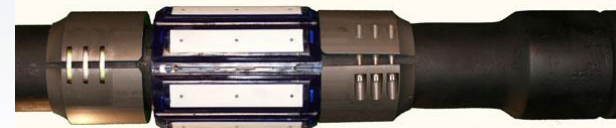
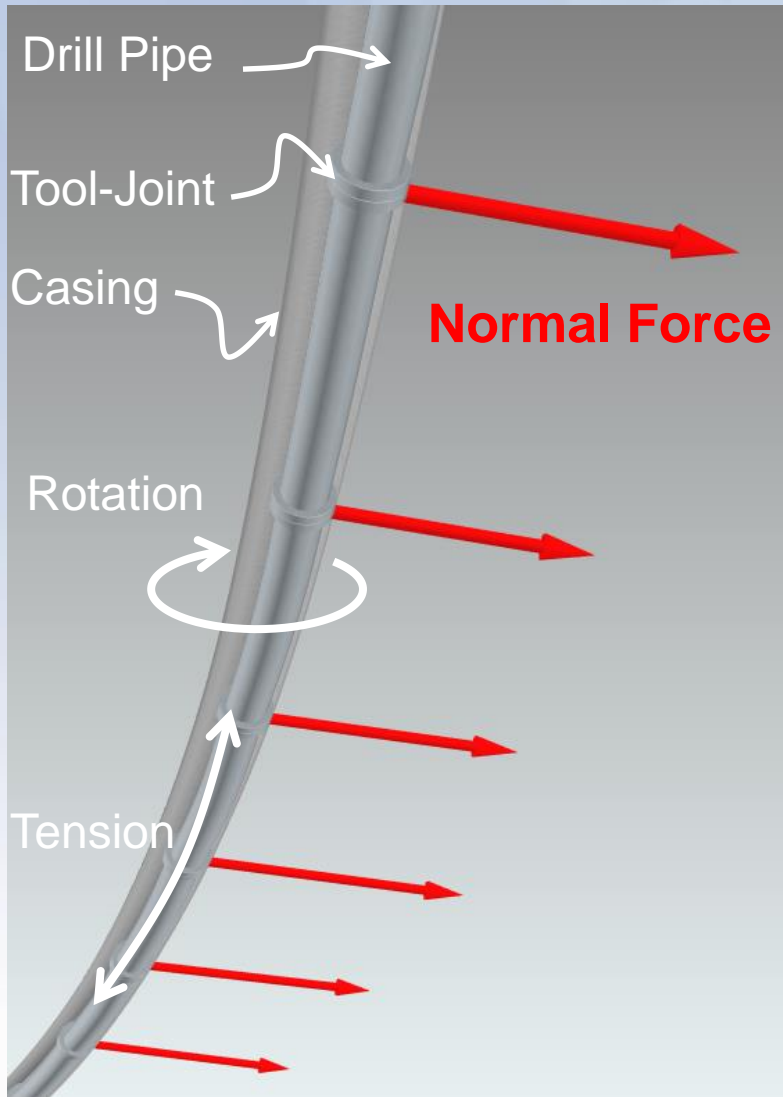


Tool Joint Wear

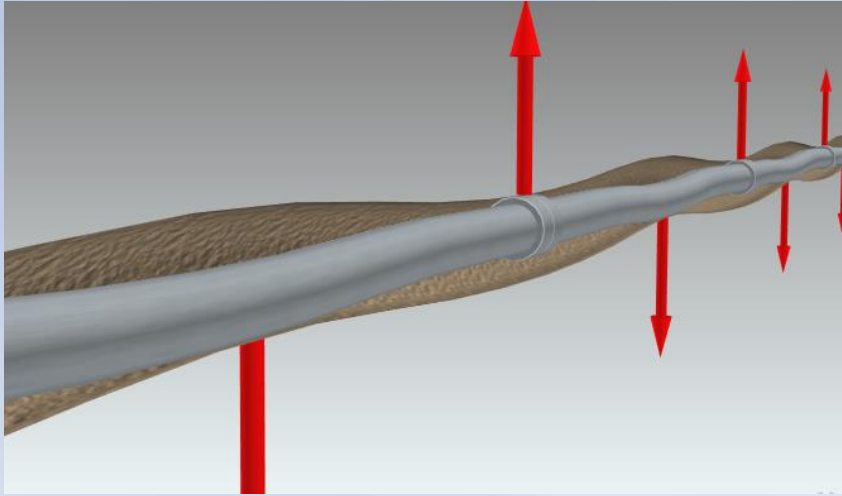


Problem Statement

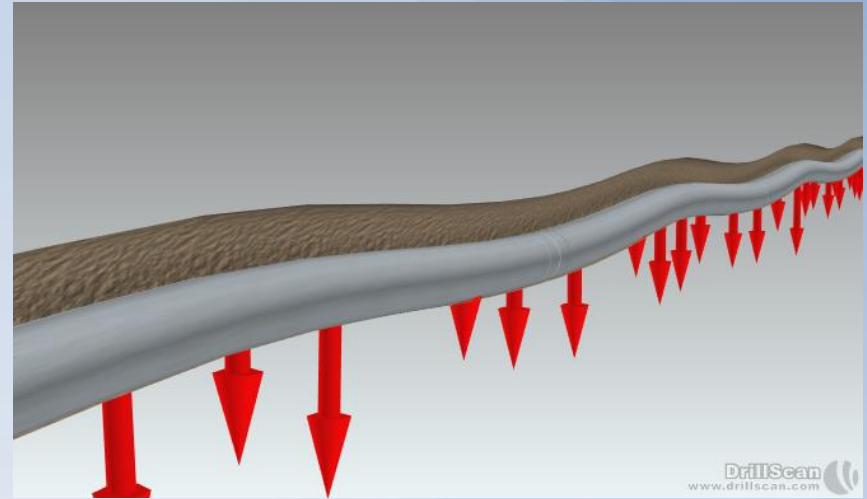
- **Factors affecting Casing Wear:**
 - **Contact Force**
 - Dog Legs in shallow parts
 - High Tension (higher contact force)
 - ROP (increasing contact time)
 - Operations (Rot. Off Bottom, Back Reaming)
 - Hard Banding (Wear Factor)
 - Mud lubricity
 - Drill Pipe Protectors



Contact Force Calculation



Stiff-String



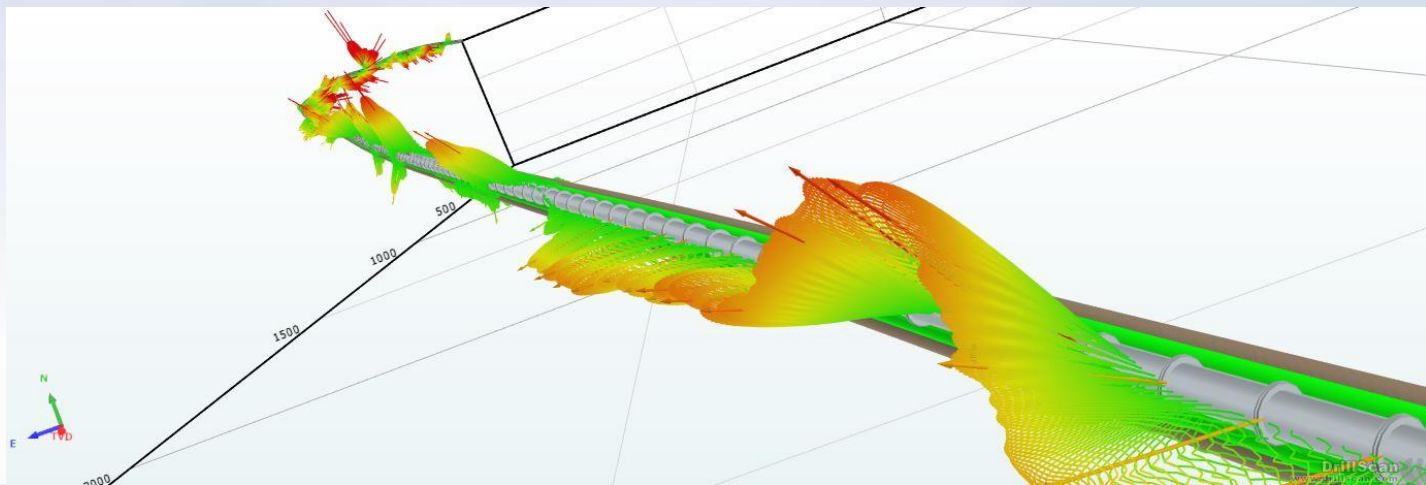
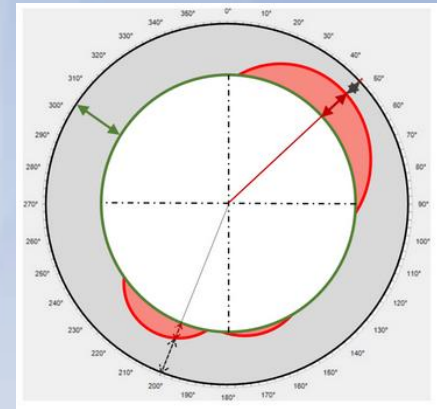
Soft-String

More Accurate Contact Force Calculation with Stiff-String

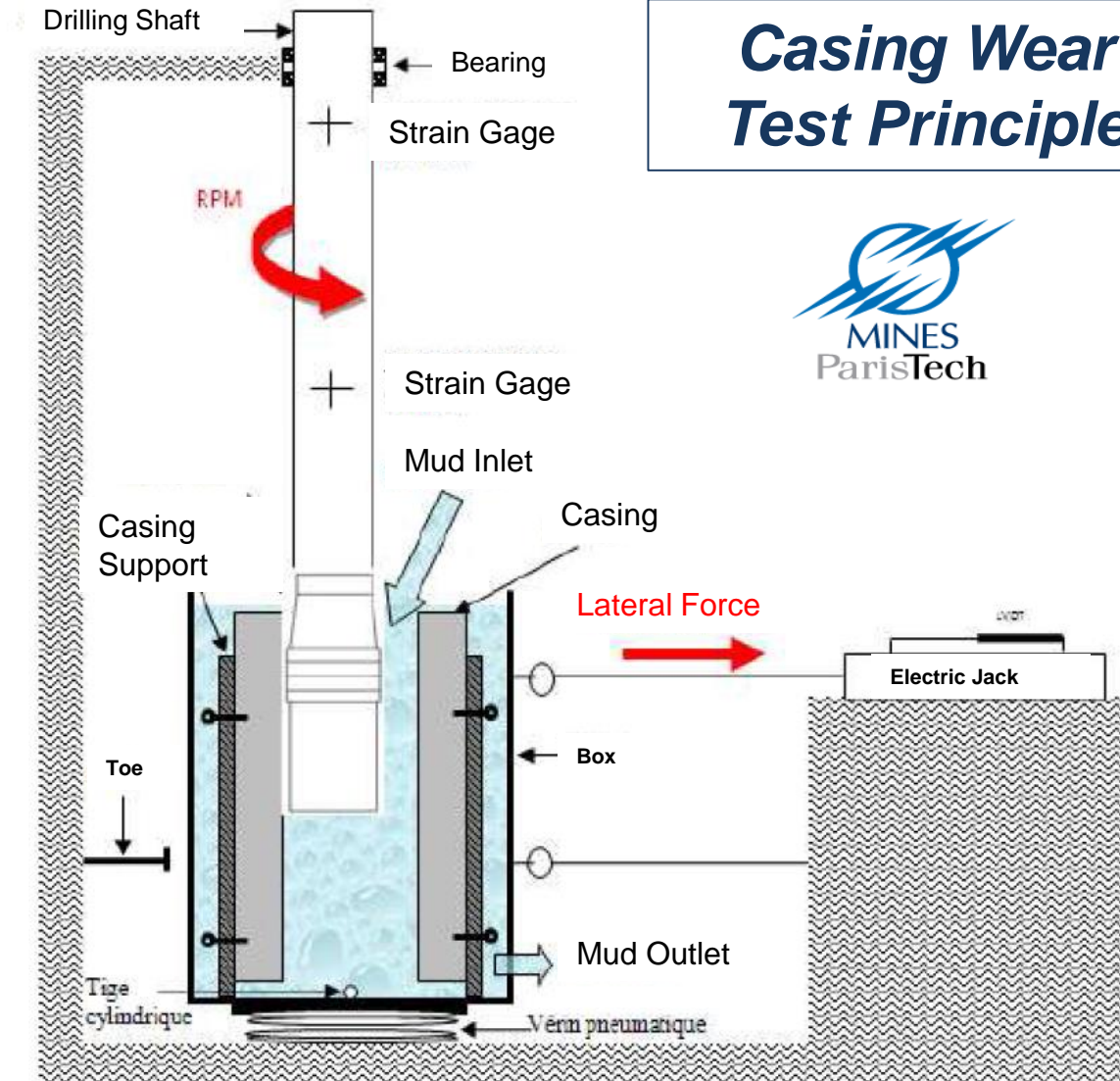
>> More accurate Casing Wear

New Casing Wear Model

- Stiff-string calculation with Contact Force Calculation
- 3D orientation of Contact Force & Wear
- Accurate Tool-Joint vs Body Contact Force
 - Wear Factor for TJ
 - Wear Factor for Body
- Realistic Dog Leg Effect (even Micro Dog Leg)
- Effect of the range of DP (Range 2 vs Range 3)
- Linear & non-linear Wear model



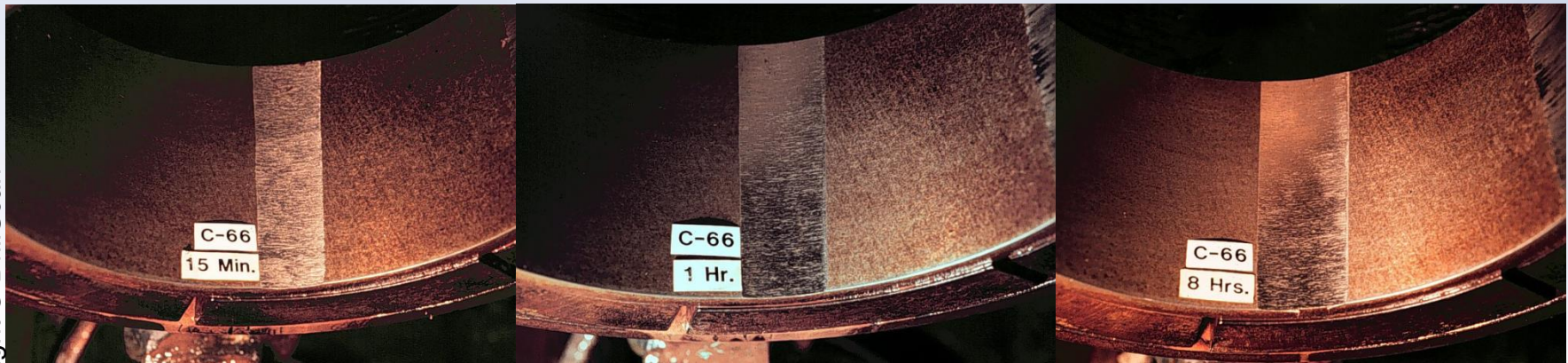
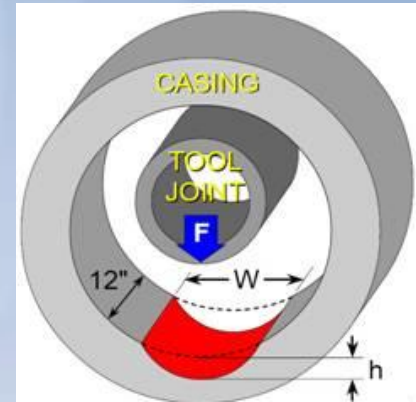
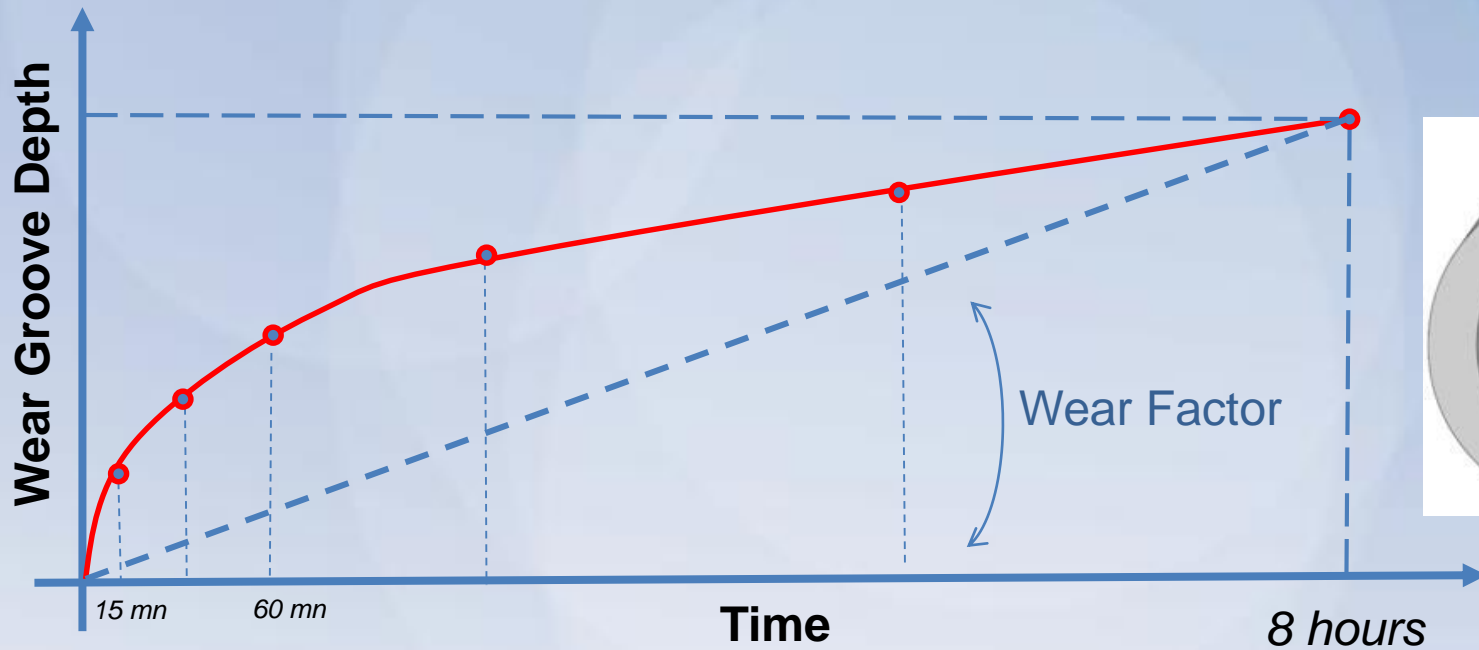
Casing Wear Test Principle




Casing Wear Test



Casing Wear Test



- R&D Project with  
- Casing Wear Tests in the Lab (API Standard 7 CW)
- Casing grade = L80, T95 & Q125
- 6 types of Hardbanding
- Effect of RPM and Side Force Studied

No Hardbanding



Type 1



Type 2



Type 3



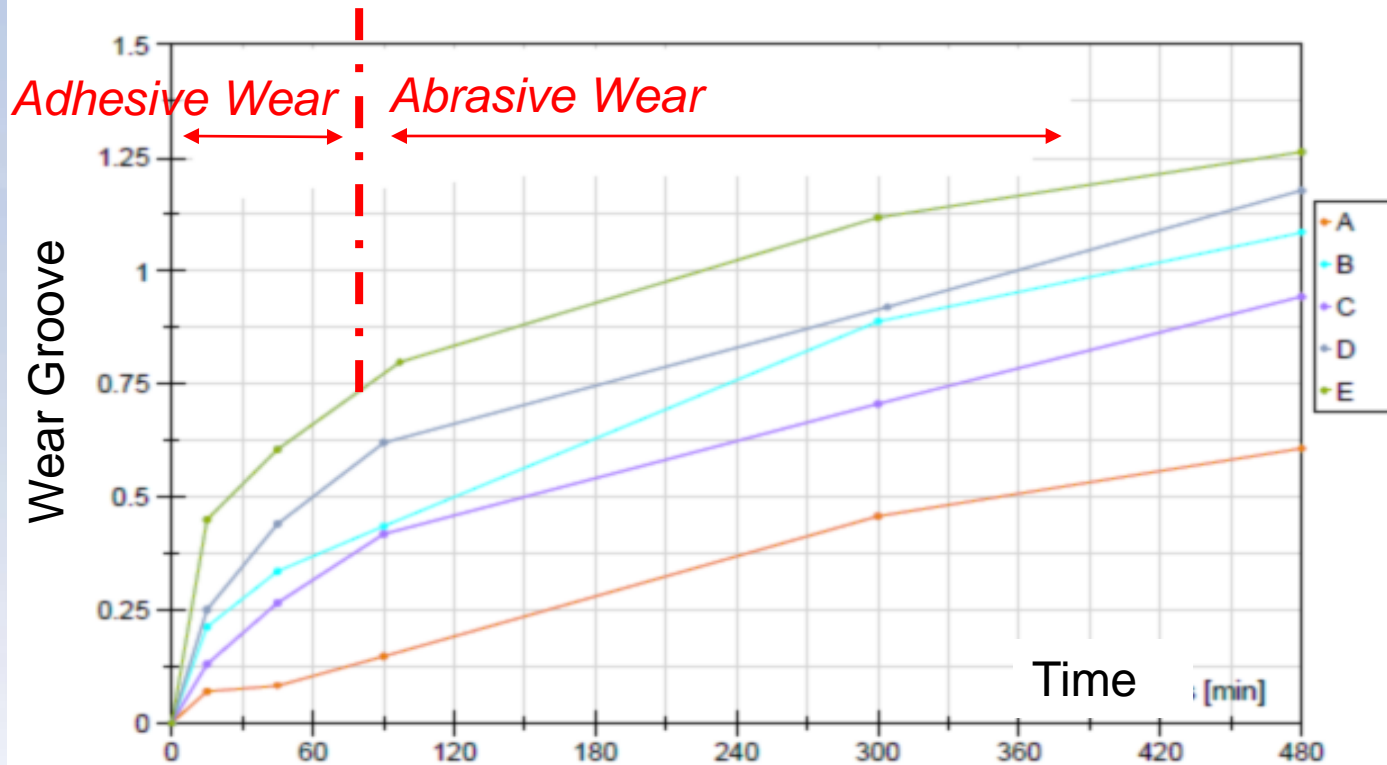
Type 4



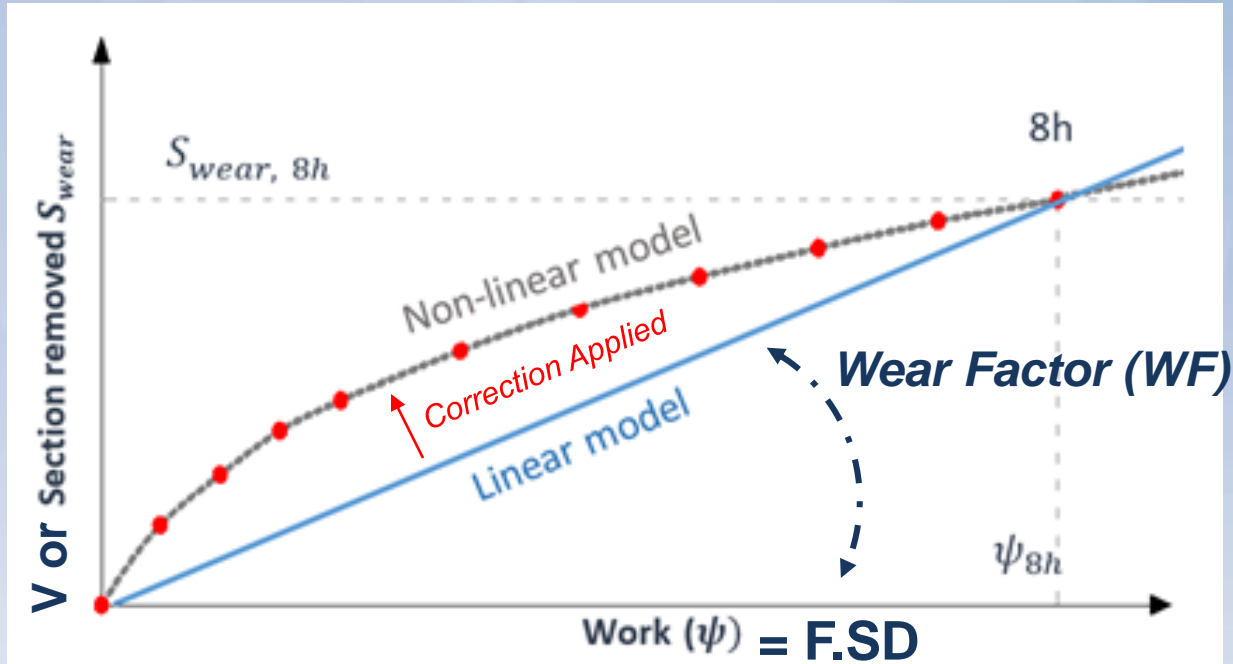
Type 5



Example of tests for 5 hard-bandings



- **Non-linearity observed**
- **Significant differences between hard-banding**
- **Slight differences with DEA42 project**



Hall's linear Model (1994)

$$V = WF \cdot F \cdot SD$$

V = Volume worn per unit length

F = Contact Force

SD = Sliding Distance = f (ROP, TJ, RPM...)

***Empirical Correction
Factor applied for
Non-Linearity***

- **Advanced String/BHA modeling required to:**
 - Optimize BHA to drill smooth wellbore
 - Neutral BHA in the lateral section
 - WOB and Gage length have an effect on BUR
 - Reduce the TVD uncertainty
 - Wellbore Reconstruction
 - Continuous Survey (Measured or Calculated)
 - Better predict completion run in hole operations
 - Torque & Drag & Buckling very sensitive to Dog Legs
 - New Buckling Severity Index to better predict the occurrence of Buckling / Failure (high stress)



*Thanks for your attention.
Any questions ?*

